

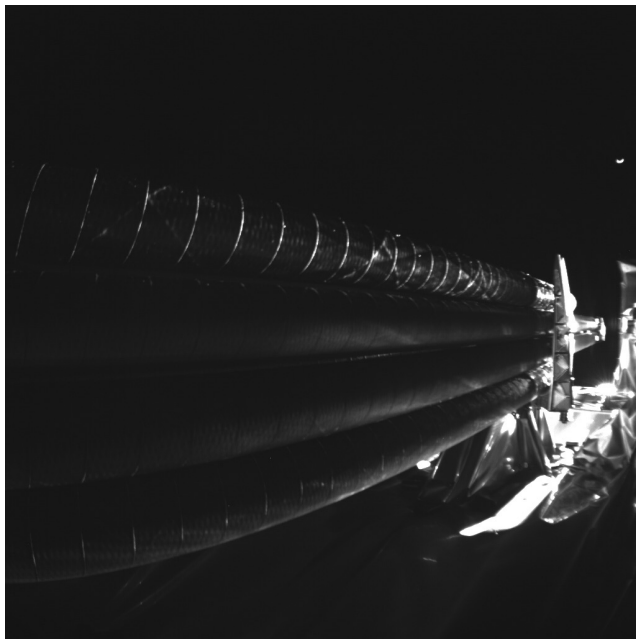
# Introduction to the Design of Space Mechanisms

Mini Project  
Mechanism Concept  
Antenna deployment

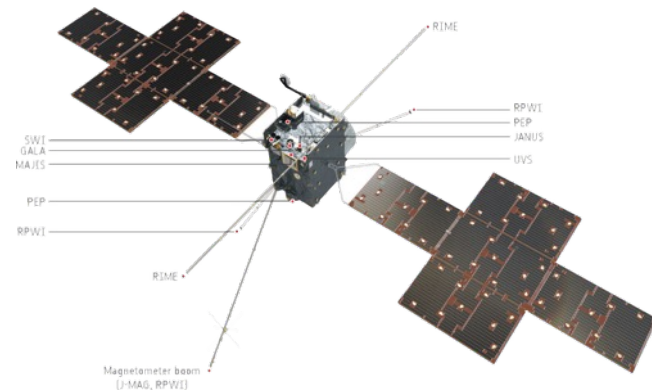
Gilles Feusier

# ESA's JUICE (Jupiter Icy Moons Explorer) - issue

## Radar for Icy Moons Exploration (RIME)



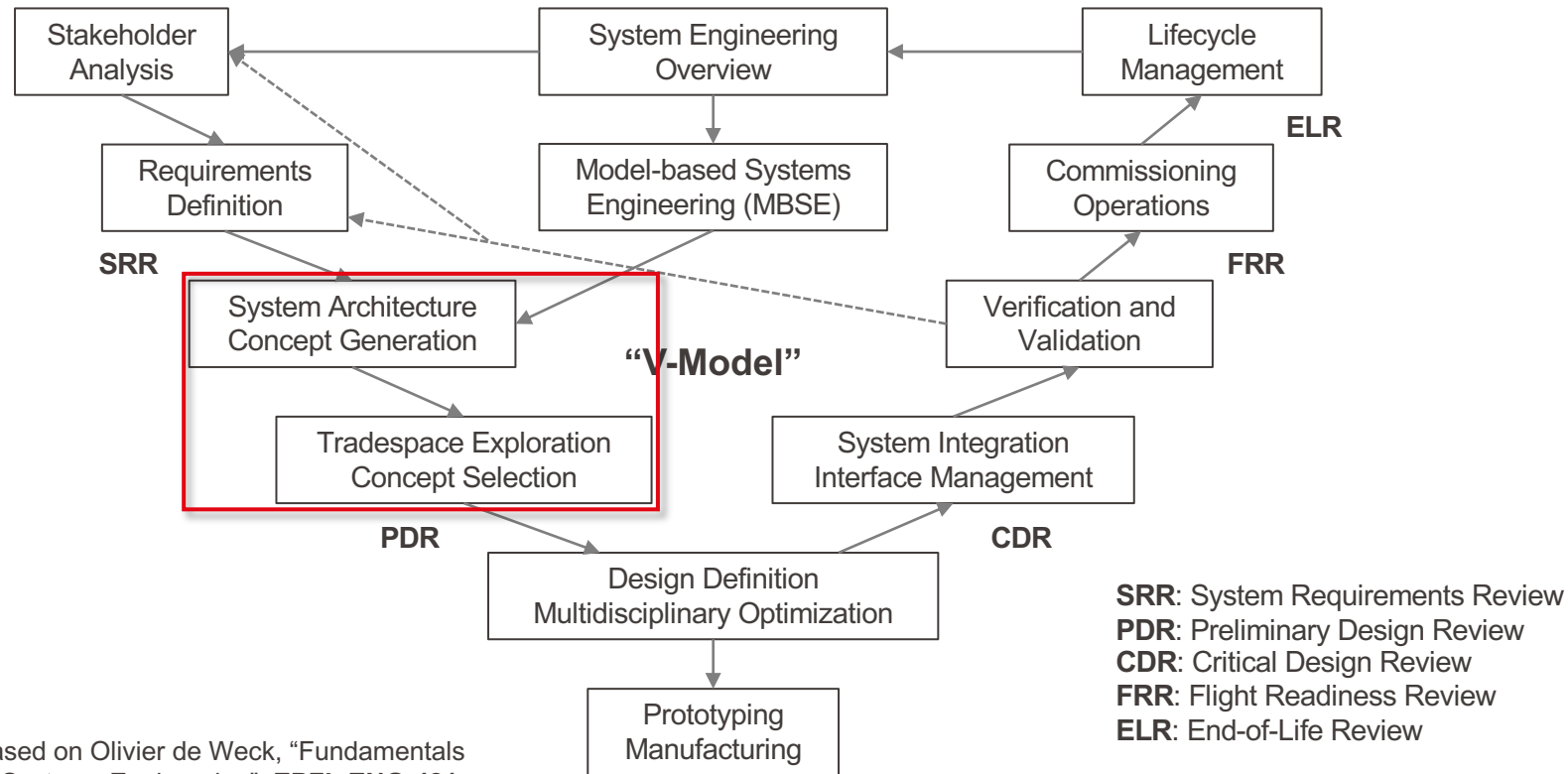
Source: ESA



Source: ESA/ATG medialab

- **17 April 2023** (three days after the launch): deployment begins
- **12 May 2023**: RIME antenna breaks free

# The “V-Model” of Systems Engineering



Based on Olivier de Weck, “Fundamentals of Systems Engineering”, **EPFL ENG-421**

Note: reviews acc. to ECSS-M-ST-10C Rev. 1

# Main Functions

## [4.2.3] Mechanical Interfaces

*Attach the payload (boom and antenna) to the S/C*

### [4.1.1.1] Deployment function

*The mechanism shall deploy an antenna (cf. Figure 1) from its stowed launch configuration to its deployed configuration*

### [4.1.1.2] The mechanism shall have one degree of freedom only.

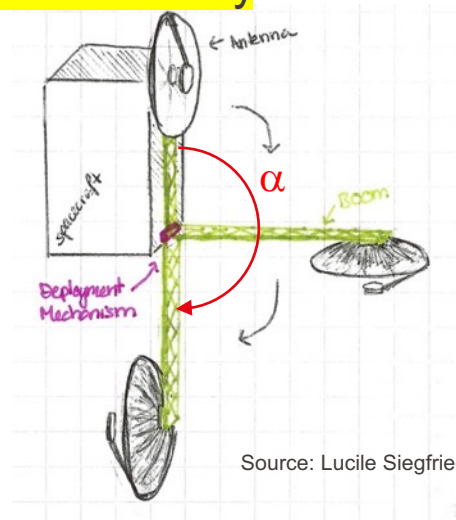
### [4.2.4.1] Angular Range of the Deployment

*The angular range of the deployment performed by the mechanism shall be  $[0^\circ; 180^\circ]$  (angle  $\alpha$  on Figure 1)*

## ■ Other functions

### [4.1.1.3] Position measurement function

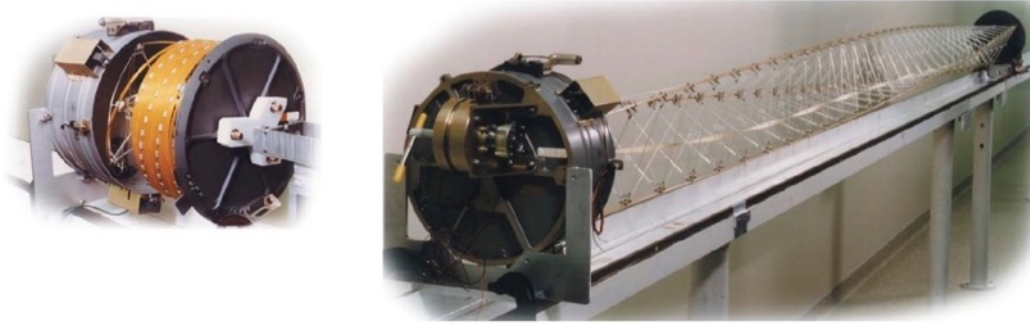
### [4.1.1.4] Position holding function



Source: Lucile Siegfried



# Other possible architecture (not compliant to req.)



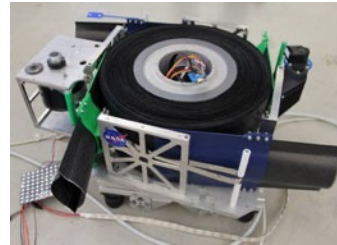
Source: N. Grumman, AstroMast Data-Sheets DS307 and DS407, 2010



Source: Sándor Kabai & Szaniszló Bérczi/  
Wolfram Demonstration Project



Source: NASA/DLR



# Critical Requirements (acc. to G. Feusier)

- [4.2.3.7] Allowed Volume
- [4.2.3.11] Mass
- [4.2.4.5] Angular Position Holding (not powered)
- [4.2.4.7] Telemetry
- [4.2.7.1] Reliability
- [4.2.9.3] Passive Holding redundancy

## Ambiguous/Missing

- [4.2.4.1], [4.2.4.5] Deployment angle: any angular position or fixed angle?
- Missing: accuracy of deployment angle ( $\alpha$ ). Only telemetry is specified.
- [4.1.2.3] Reversibility: in-orbit?
- Cable harness to antenna: missing

# Environmental Requirements

**Severe, but quite standard:**

- [4.3.1.6] Shocks (... 100g/1300Hz, 700g/2000Hz, ...)
- [4.3.1.4] Random vibrations (23.4g<sub>rms</sub>)
- [4.3.2.11] Space thermal environment (qualification: +90 to -30°C)

Other requirements are considered to be standard, including radiation and cleanliness requirements, ...

# Main Functions

- I/F
  - Fixed to S/C structure
  - Boom with antenna fixed to the mechanism
  - Electrical harness to antenna: TBD
- Environment
  - Withstand launch loads
  - ...
- Operations
  - Non-operational: Stowed (**secured**) position during storage, launch...
  - Operational: deployment within defined constraints (speed, acceleration, range) => **rotation**
  - Passive: **maintain** the defined (pos. TBD) position during antenna operations
- Telemetry
  - Angular position

# Selection of Components

- Rotation =  $180^\circ$
- Position holding (passive, redundant)
- Telemetry
- Structure



# Selection of Components

- Rotation =  $180^\circ$ 
    - Ball-bearing
  - Rotation
    - Actuator: TBD<sup>(\*)</sup>
  - Position holding
    - Latch and end stops?
    - Brake?
    - Nothing else?
  - Telemetry
    - Angular encoder
    - Micro-switches (TBC<sup>(\*)</sup>): status monitoring
  - Structure
- Selected concept depends on deployment angle and reversibility

(\*) TBD: To Be Defined, TBC: To Be confirmed

# Deployment performances

- Range:  $[0;180]^\circ$  [4.2.4.1]
- Maximum rate:  $6^\circ/\text{s}$  [4.2.4.2]
- Maximum acceleration:  $1 \text{ rad}/\text{s}^2$  [4.2.4.4]
- Maximum duration: 40s [4.2.4.3]

$$\alpha(t) = \alpha_0 + \omega t + \frac{1}{2} \dot{\omega} t^2$$

- *Acceleration phase:  $\dot{\omega}_{init} > 0$*
  - *Constant speed phase (if any):  $\dot{\omega} = 0$ ,  $\omega = \omega_{constant}$*
  - ...
  - *Deceleration phase:  $\dot{\omega}_{fin} < 0$*
- Required torque ?  
 $T = I\dot{\omega}$  with  $I = 3.8 \text{ kg}\cdot\text{m}^2$  [4.2.3.5]

# Deployment performances (an example)

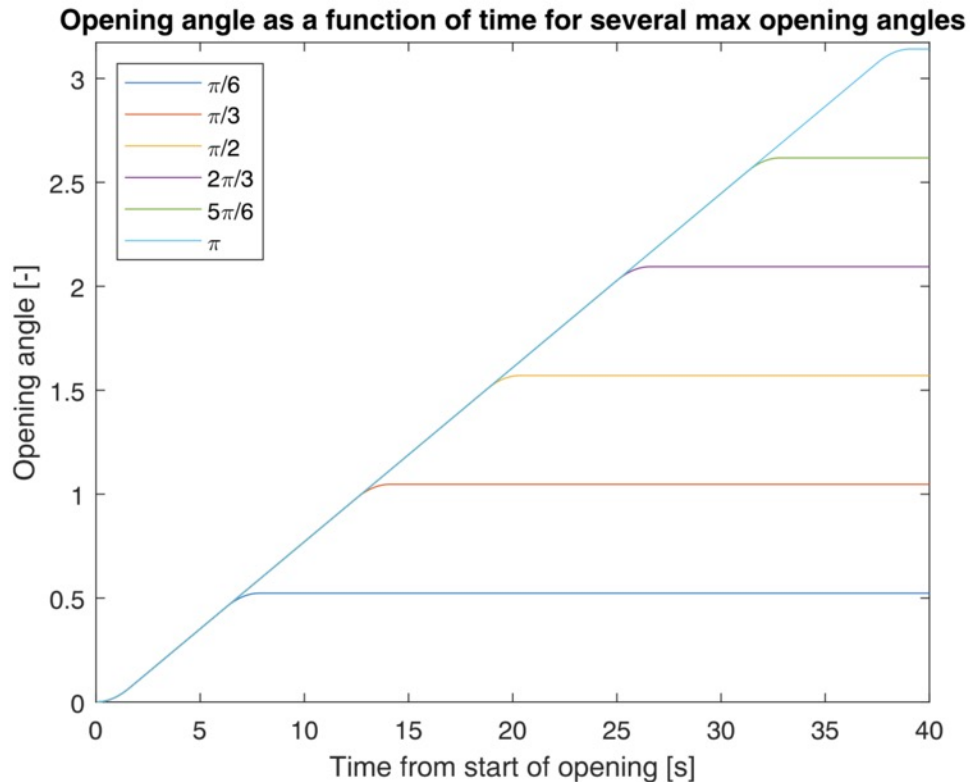


Figure 1: Antenna deployment as a function of time for various target angles.

With:

$$\omega_{constant} = 80\% \times \omega_{max}$$

$$T = 0.2 \text{ Nm}$$

$$\text{Duration} \leq 40\text{s}$$

Source: Arnaud Muller, Antoine Clout

# Deployment performances

- Torque margin
  - ECSS-E-ST-33-01C Rev.2 - Mechanisms

Table 4-2: Minimum uncertainty factors for actuation function

Resistive torque or force contributors	Symbol	Theoretical Factor	Measured Factor
Inertia	$I$	1,1	1,1
Spring	$S$	1,2	1,1
Magnetic effects	$H_M$	1,5	1,1
Friction	$F_R$	3	1,5
Hysteresis	$H_Y$	3	1,5
Others (e.g. Harness)	$H_A$	3	1,5
Adhesion	$H_D$	3	3

$T_D$  : inertial resistance torque  
 $T_L$  : deliverable output torque  
 (if specified by customer)

$$T_{min} = 2 \cdot [1.1 \cdot I + 1.2 \cdot S + 1.5 \cdot H_M + 3 \cdot F_R + 3 \cdot H_Y + 3 \cdot H_A + 3 \cdot H_D] + 1.25 \cdot T_D + T_L$$

- Example
  - Inertia (e.g.):  $T_D = 0.2 \text{ Nm} \Rightarrow 1.25 \times 0.2 = 0.25 \text{ Nm}$
  - Friction:  $0.02 \text{ Nm}$  per duplex bearing  $\Rightarrow 2 \times 3 \times (2 \times 0.02) = 0.24 \text{ Nm}$
  - **Total: 0.49 Nm** (no harness, no spring, only bearing friction ...)

<http://tiny.cc/EE580Q04>





# Actuator

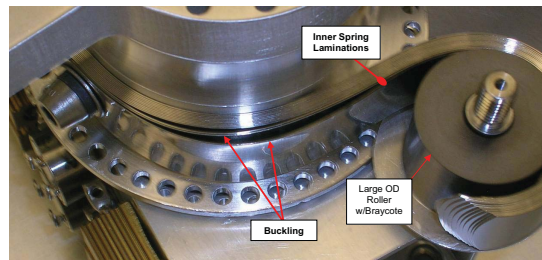
- Brushless DC motor
- Stepper motor
- Sealed brushed motor
- ❖ Gear box
- ❖ Complex control electronic (brushless motor, stepper motor)

- Spring (passive)
  - Helical torsion spring
  - Constant torque spring
  - ...

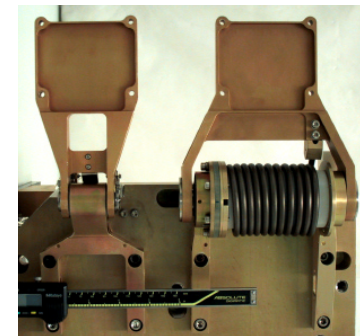
- Hold-down device (e.g. pin-puller, shall be resettable)
- Damper (deployment rate control)
- Resettable: on ground, not in-orbit



Source: C. Boesch et al., Proceedings of the 13th European Space Mechanisms and Tribology Symposium, 23-25 September 2009



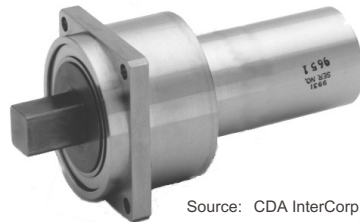
Source: J.A. Johnson, Proceedings of the 39th Aerospace Mechanisms Symposium, NASA Marshall Space Flight Center, May 7-9, 2008.



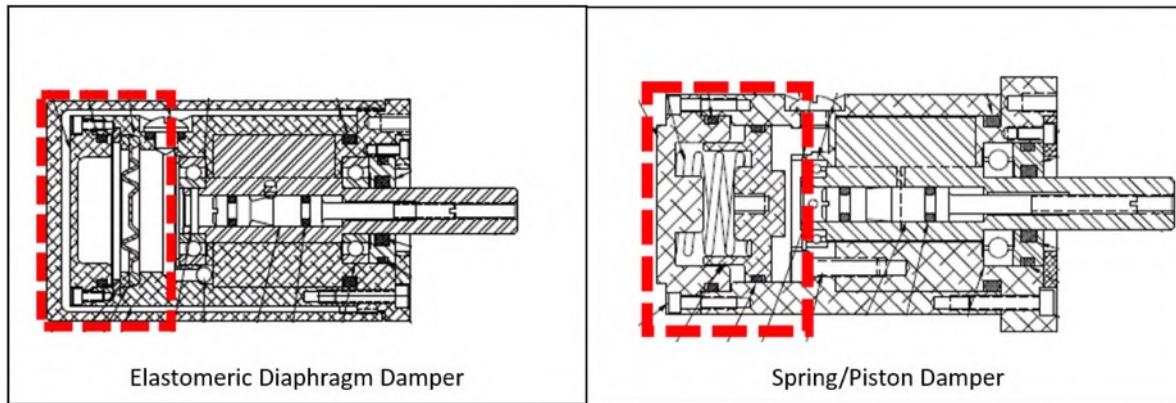
Source: J.I. Bueno et al., Proceedings of the 9th European Space Mechanisms and Tribology Symposium, 19-21 September 2001

# Spring Actuator: acceleration/speed control

- Damper:
  - Eddy current damper
  - Viscous damper



Source: CDA InterCorp, USA



Source: P. Lytal et al., Proceedings of the 44th Aerospace Mechanisms Symposium, NASA Glenn Research Center, May 16-18, 2018



Source: W. Mitter et al., Proceedings of the 13th European Space Mechanisms and Tribology Symposium, 23-25 September 2009

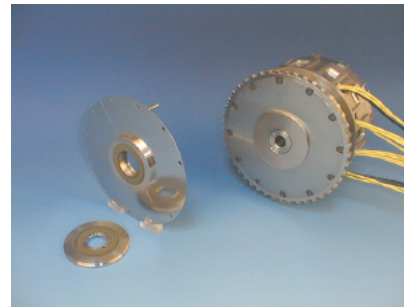
Viscous: Flight heritage, but for much larger torque. Not a commercial product. Depends also on full mechanism characteristics. Depends on temperature (thermal control required).

Eddy current: commercial products available

Very critical  
Early stage analysis. BBM testing?

Performances to be assessed.  
Includes gear box?

Source: F.C. Baker et al., Proceedings of the 8<sup>th</sup> ESMATS, 29 Sept.-1 Oct. 1999

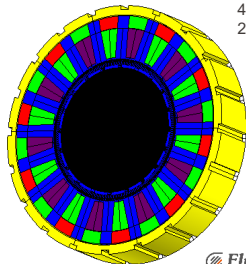


- Mechanical brake
  - Shall operate under vacuum and under air
    - Change of tribological characteristics, non-linear
    - Running-in when changing environment?
  - Requires an actuator (contact pressure, braking torque force, power)
  - Behavior under vibrations?
  - Size
  - Monitoring the status?
  - No (limited) commercial product available



Source: T. Hopper et al., Proceedings of the 43<sup>rd</sup> AMS, 4-6 May 2016, NASA/CP-2016-219090

- Other option: reluctance brake
  - Passive or Active

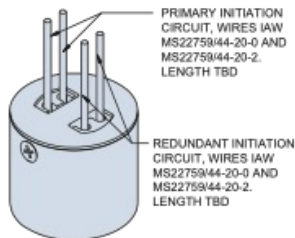


## ELECTRICALLY REDUNDANT HOLD DOWN RELEASE MECHANISM, MEDIUM DUTY



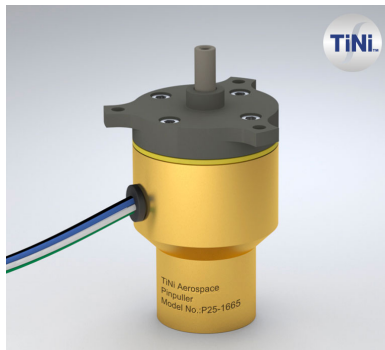
Source: Glenair, Inc

How To Order		
Sample Part No.	061	-007
Basic Part No.	Light/Medium Duty HDRM	
Dash No.	Redundant Circuit	



Reset requires the replacement of the initiator

**NOK**  
(reversibility?)



Source: Ensign-Bickford Aerospace &amp; Defense Company

## P5 Details

Mass:	1.06 oz [30 g]
Power:	1.25 W @ 0.4 A
Operational Current:	0.4 to 1.5 A
Resistance:	7.7 ± 0.5 Ω @ 23°C
Pull Force:	5 lbf [22.2 N] MIN
Pull Stroke:	0.250 in [6.3 mm] MIN
Axial Load (Actuation):	2 lbf [8.9 N] MAX
Side Load (Actuation):	10 lbf [44.5 N] MAX
Side Load (Non-Actuation):	100 lbf [444.8 N] MAX
Function Time:	130 msec. MAX @ 0.5 A (23°C)
Reusable:	By Manual Reset
Life:	100 Cycles MIN
Operational Temperature:	-65° C to +70° C
Size (max):	D31.75 x 41.275 mm

Can be reset  
However  $T_{op}$  range  
not OK

Check if  $T_{op}$   
can be  
increased

# Angular encoder, switches

## Potentiometer



Source: Betatronix



Source: Novotechnik

Resolution: better than  $0.01^\circ$   
(*even better than  $0.007^\circ$* )  
 $\Rightarrow 0.17 \text{ mrad}$

## Optical encoder



Source: Codechamp

Resolution: 15bits:  $\sim 0.011^\circ$   
(*can reach 23bits, i.e.  $0.07 \cdot 10^{-3} \text{ mrad}$* )  
 $\Rightarrow 0.19 \text{ mrad}$

## Micro-switch



Source: Honeywell

## Reed switch



Source: Standex Electronics, Inc.

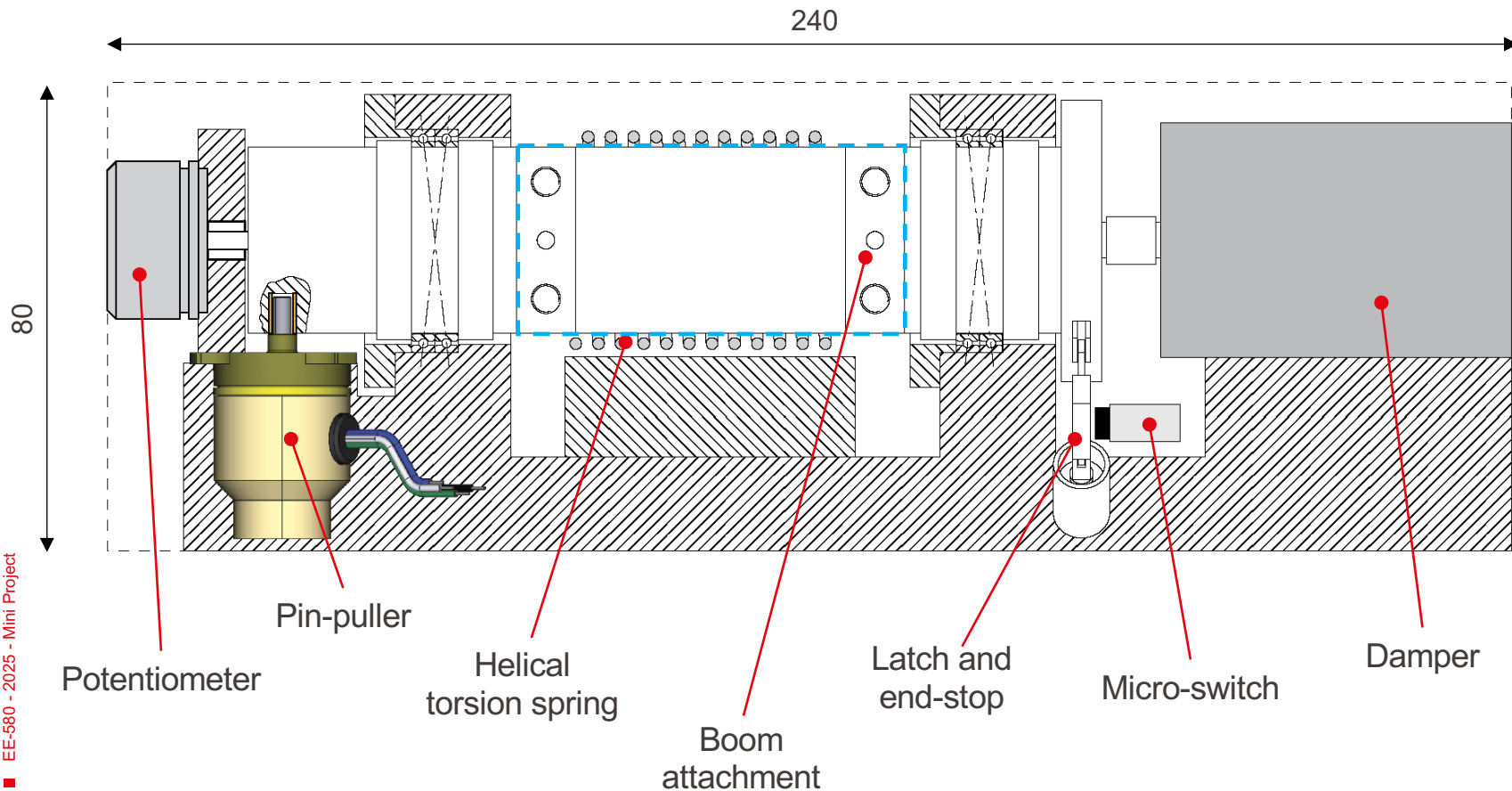
Requirement [4.2.4.7]:  $< 0.2 \text{ mrad}$



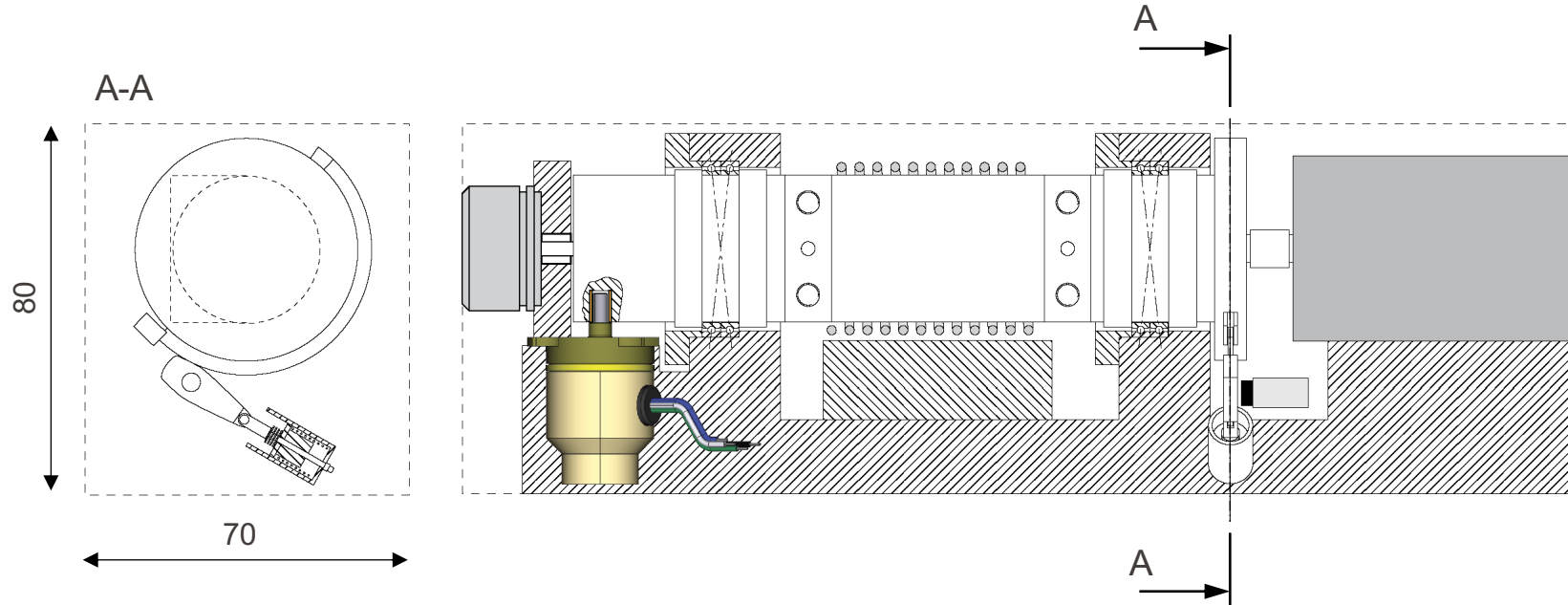
- Pugh matrix

		Concept 1	Concept 2	Concept 3	Concept 4
Criteria 1	Mass	+	Datum	-	+
Criteria 2	Size	-		+	-
Criteria 3	Control	S		+	-
Criteria 4	Power	S		+	S
...	...				
$\Sigma+$		1		3	1
$\Sigma-$		1		1	2
$\Sigma S$		2		0	1
$\Sigma+ - \Sigma-$		0		2	-1

# Concept with spring actuator



# Concept with spring actuator

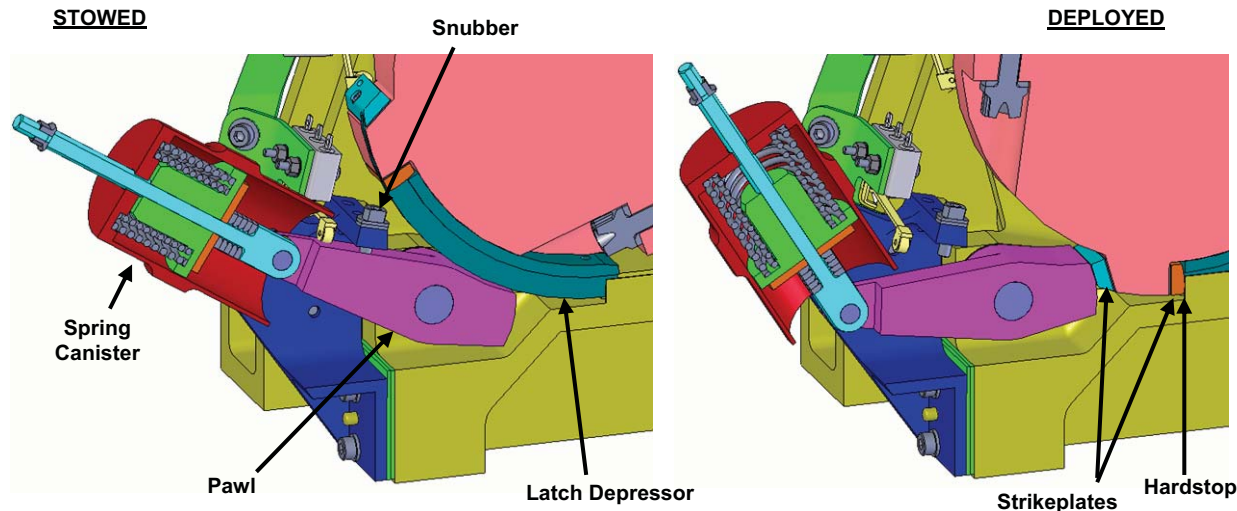


As low force is required, latch maybe simplified

# Concept with spring actuator

- Latch concept (example)

From Joel A. Johnson “*Development of the Aquarius Antenna Deployment Mechanisms and Spring/Damper Actuator*”, Proceedings of the 39th Aerospace Mechanisms Symposium, NASA Marshall Space Flight Center, May 7-9, 2008.



# Power budget: selection of cables

Components	Model	# wires	Max $I_r$ [A]	AWG	Length [m]	Total mass [g]
Pin puller	EBAD P5	4	1.5	AWG26	0.2	1.6
End-switch	Honeywell HM	6	<0.001	AWG28	0.2	1.6
Potentiometer	Novotechnik P2500	3	<1·10 <sup>-6</sup>	AWG28	0.2	0.8
<b>Total</b>		$N_{nom} =$ <b>13</b>				<b>4</b>

Single wires acc. to ESCC 3901 001

## Cable derating: ECSS-Q-ST-30-11C Rev 1 Derating - EEE components

Maximum current for single wire

AWG26      2.5 A

AWG28      1.5 A

**Bundle factor  $K$**  for  $N$  wires (derated allowable current:  $I_{derated} = K \cdot I_r$ ):

$N = \lceil 0.5 \times N_{nom} \rceil$  for cold redundancy wires or not in the same bundle  $\Rightarrow N = 7$

$3 < N \leq 7$        $K = 1.01 - (0.07 \times N)$        $\Rightarrow$        **$K = 0.52$**

Derated Current:

**AWG26      1.3 A**

**AWG28      0.78 A**

$\Rightarrow$  pin puller **current shall be limited**: OK ( $I > 0.4$  A, function time)



# Mass budget

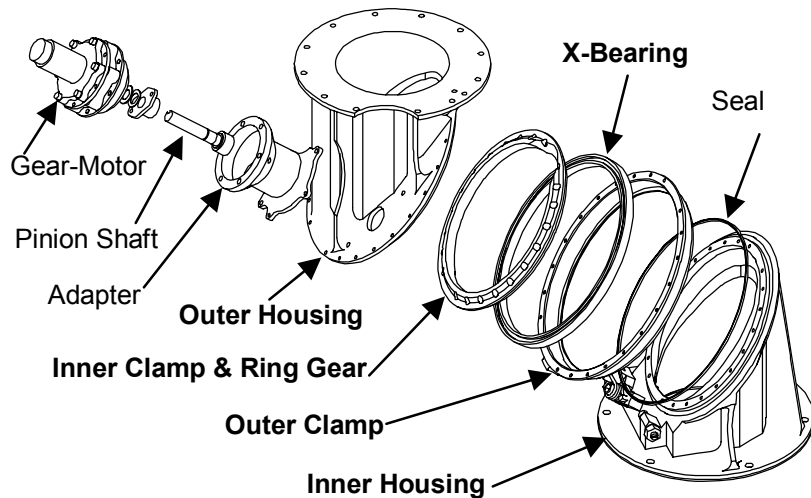
Components	Model	Quantity	Unit mass [g]	Total mass [g]
Duplex bearing	ADR WAD420	2	15	30
Damper		1	150	90
Pin puller	EBAD P5	1	30	30
End-switch	Honeywell HM	2	10	20
Spring		1	50	50
Potentiometer		1	20	20
Screws	M3x8	50	0.3	15
D-SUB 25		2	12	24
Cables				4
Total				283
Contingency 10%				28.3
<b>Total</b>				<b>311.3</b>

Available for structure and parts (incl. coupling, latch ...):  $700 \text{ g} - 311.3 \text{ g} = 388.7 \text{ g}$



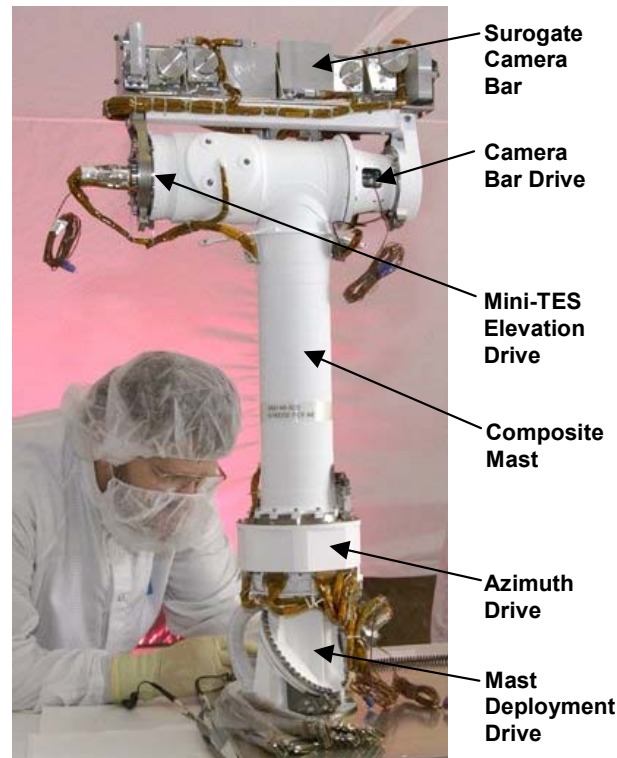
Not much. Structure shall be optimized!

# Gear-Motor concept: Mars Rover PMA



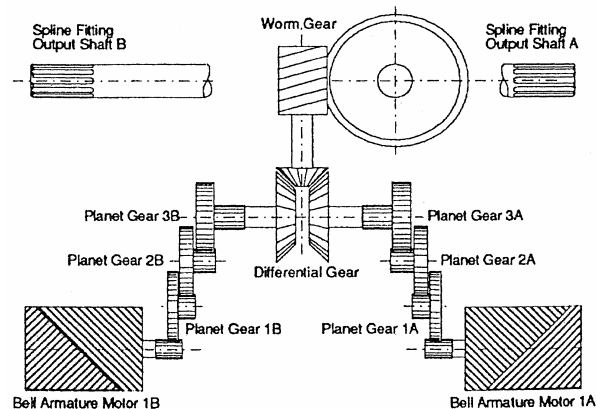
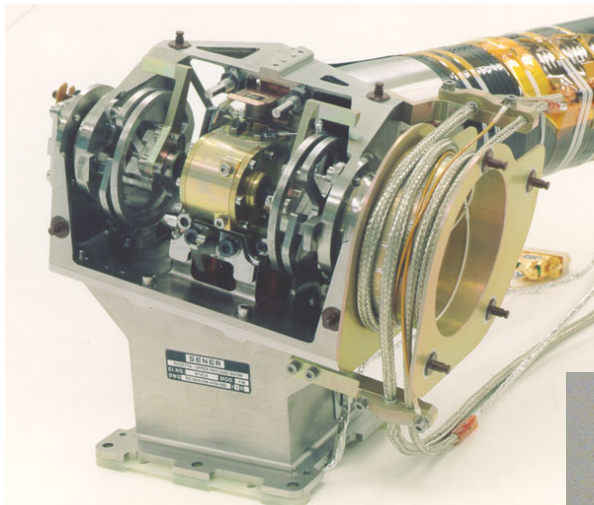
**Figure 8: Exploded view of the Mast Deployment Drive**

Source: R.M. Warden et al., Proceedings of the 37th Aerospace Mechanisms Symposium, Johnson Space Center, May 19-21, 2004.

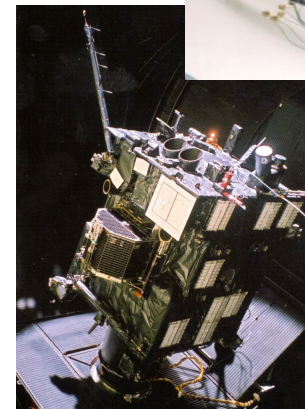
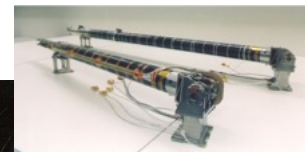
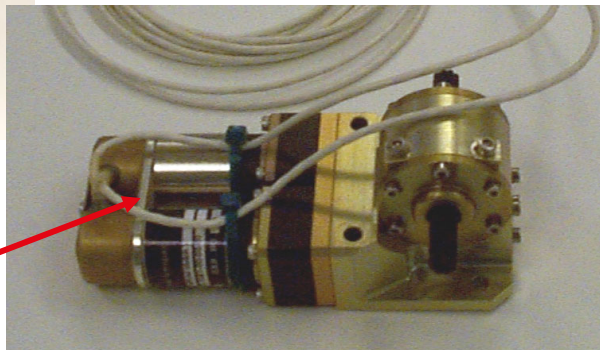


**Figure 2. PMA in the Deployed Configuration**

# Worm Gear concept: Rosetta deployable boom

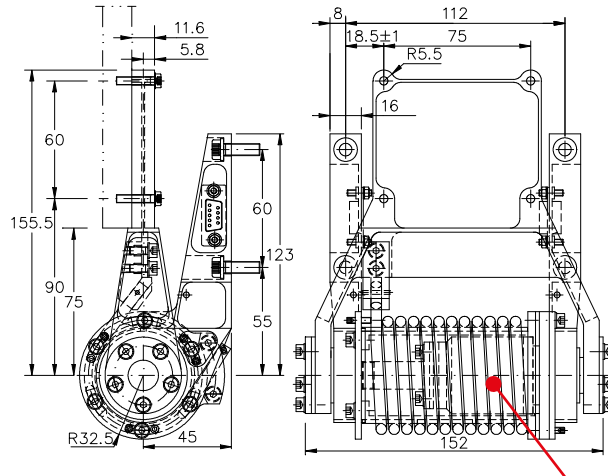


two redundant sealed  
brush motors

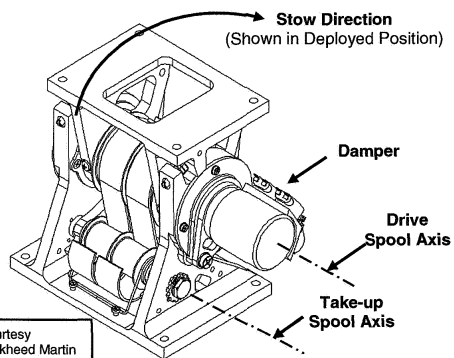


Source: J.A. Andión et al., Proceedings of the 10th European Space Mechanisms and Tribology Symposium, 24-26 September 2003

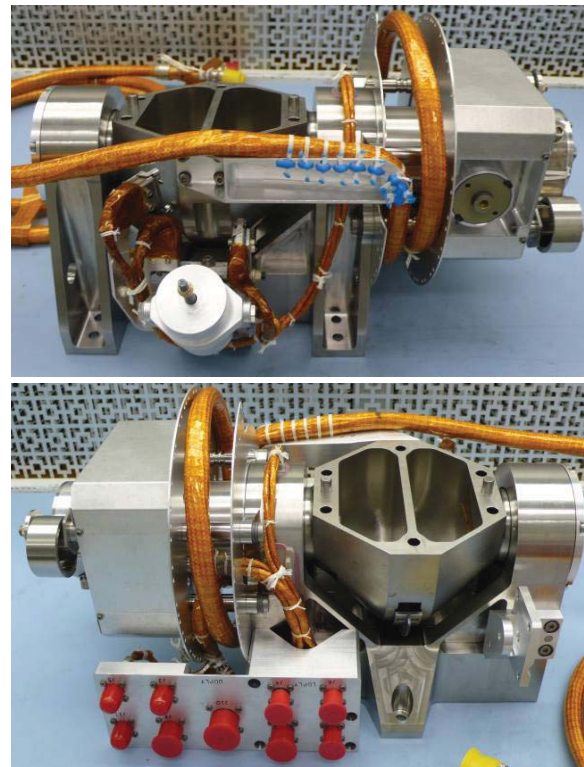
Source: J.I. Bueno et al., Proceedings of the 9th European Space Mechanisms and Tribology Symposium, 19-21 September 2001



Deployment  
Regulator

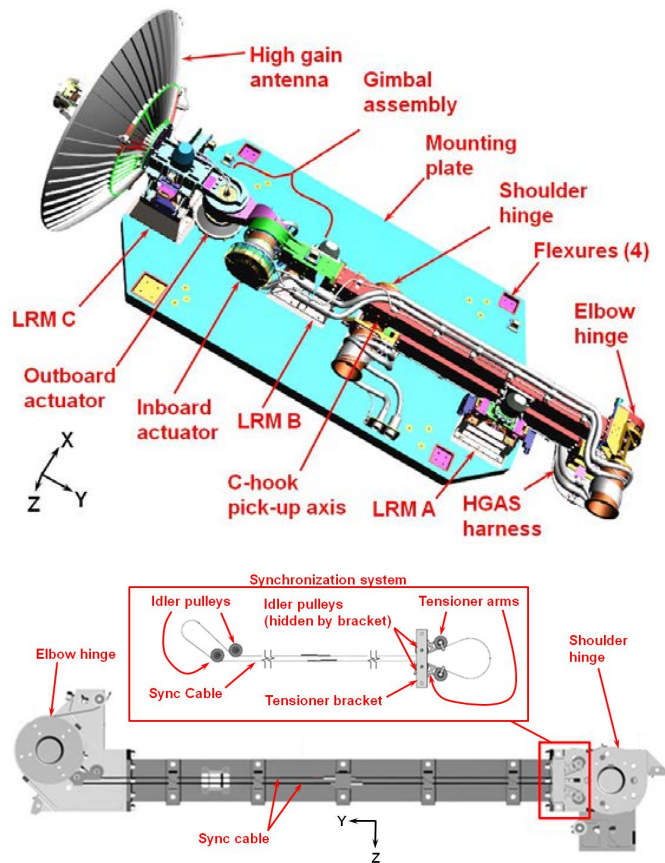


Courtesy  
Lockheed Martin  
Corp.

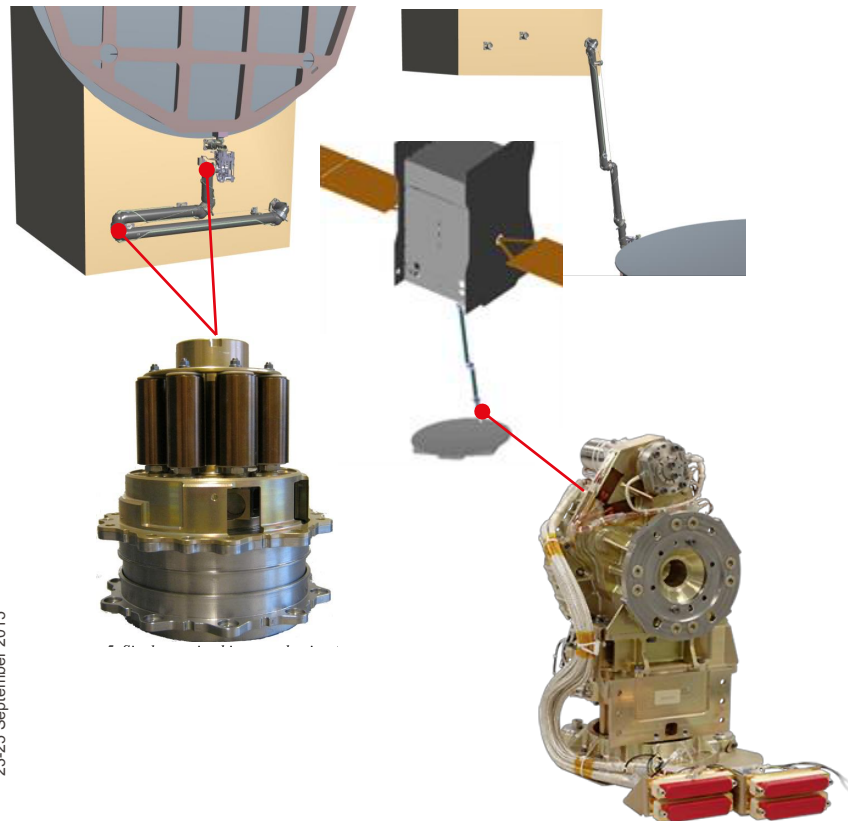


Source: J. A. Johnson, Proceedings of the 39th Aerospace Mechanisms Symposium, NASA Marshall Space Flight Center, May 7-9, 2008.

# Antenna deployment



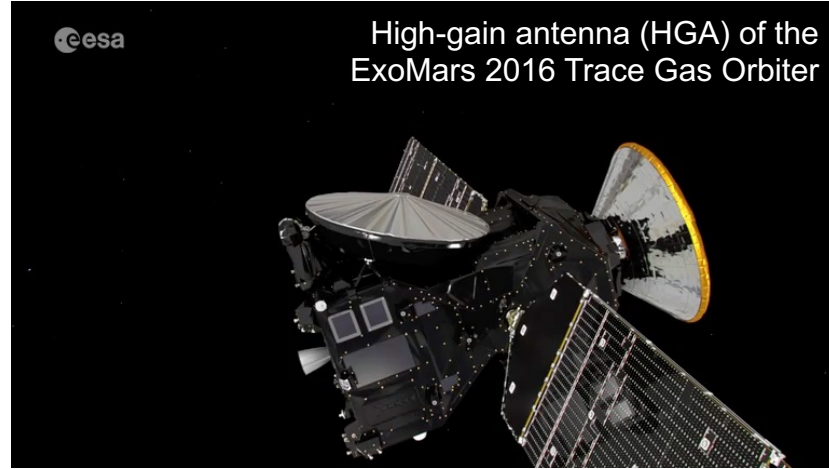
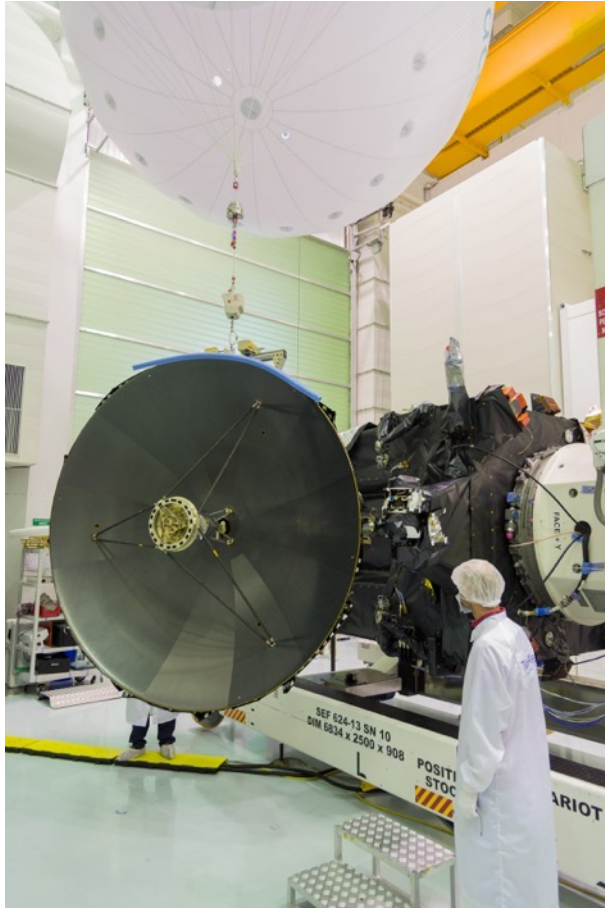
Source: F. Parong et al., Proceedings of the 42nd Aerospace Mechanisms Symposium, May 14-16, 2014



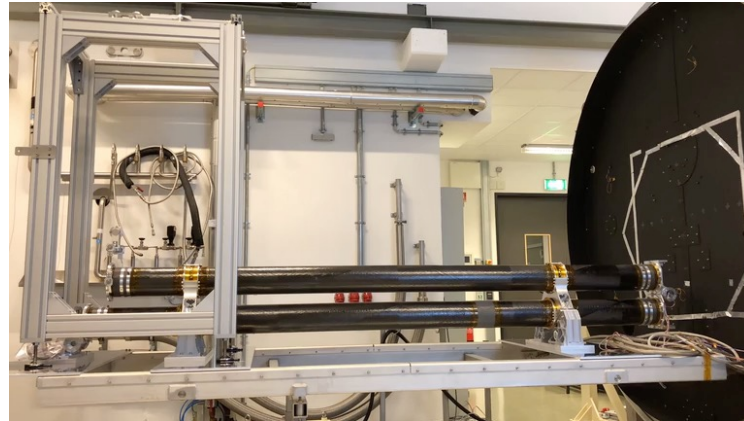
Source: M. Kroon et al., Proceedings of the 16th European Space Mechanisms and Tribology Symposium, 23-25 September 2015



Source: ESA - B. Bethge



Source: ESA/ATG medialab

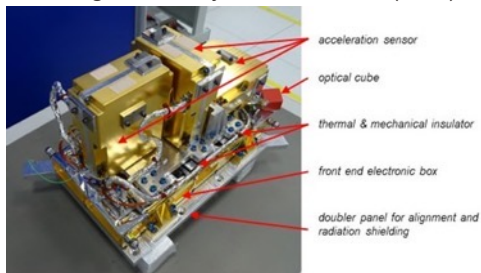


Source: ESA-A. Ihle

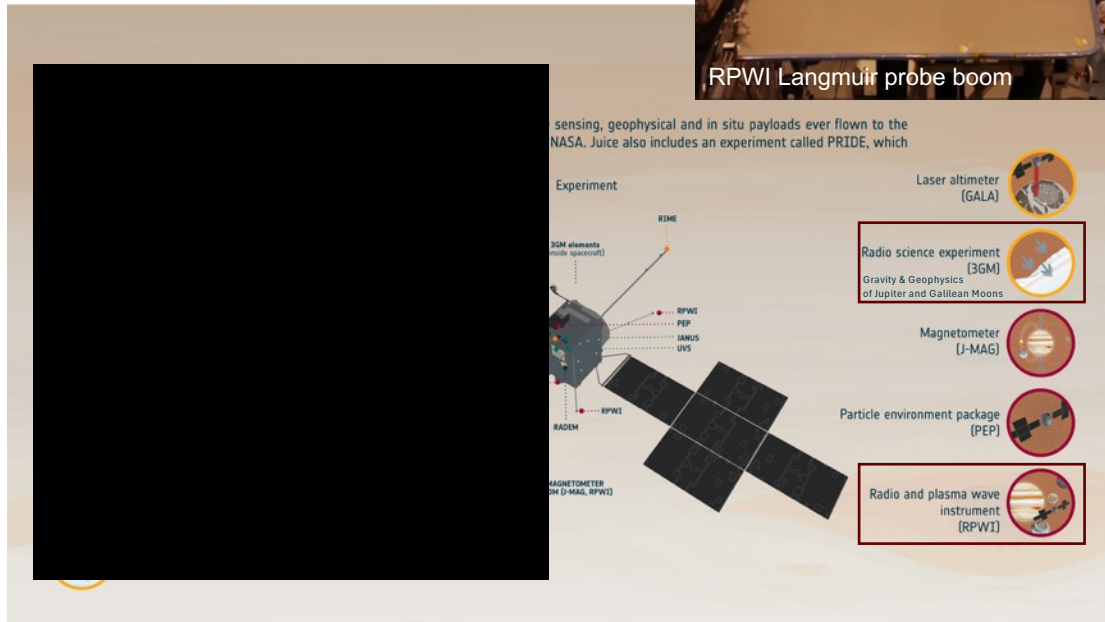
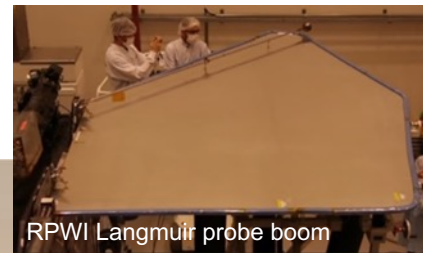
# Deployment (booms and others)

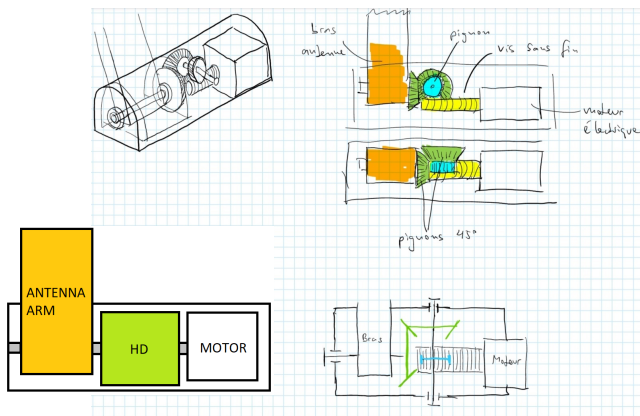
- **Indirect ways to monitor:** e.g. use of an onboard accelerometer to monitor the deployment of a boom
  - Bandwidth of the accelerometers
  - Comparison between ground and in-orbit values maybe difficult

3GM High Accuracy Accelerometer (HAA)

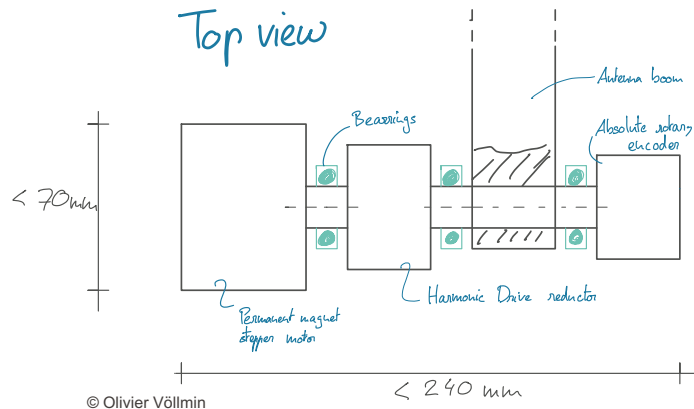


R. Le Letty et al., “Listening into the JUICE Deployments with the On-Board High-Accuracy Accelerometer”, Proceedings of the 47th Aerospace Mechanisms Symposium, NASA Langley Research Center, May 15-17, 2024

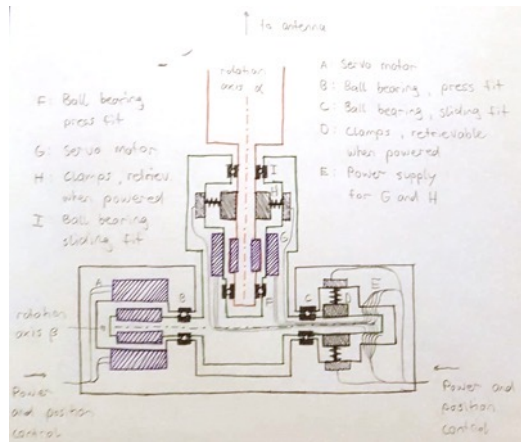




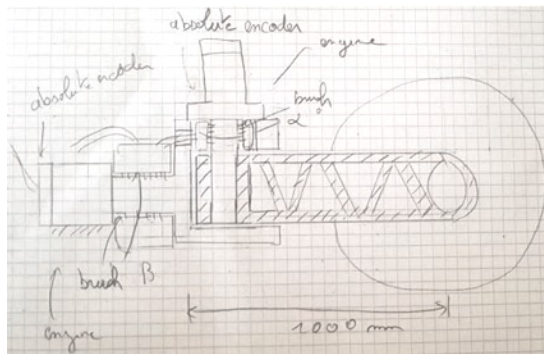
© Thomas Pfeiffer



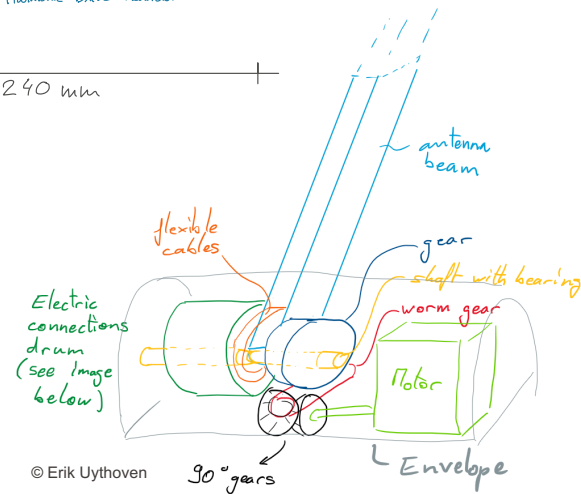
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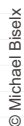


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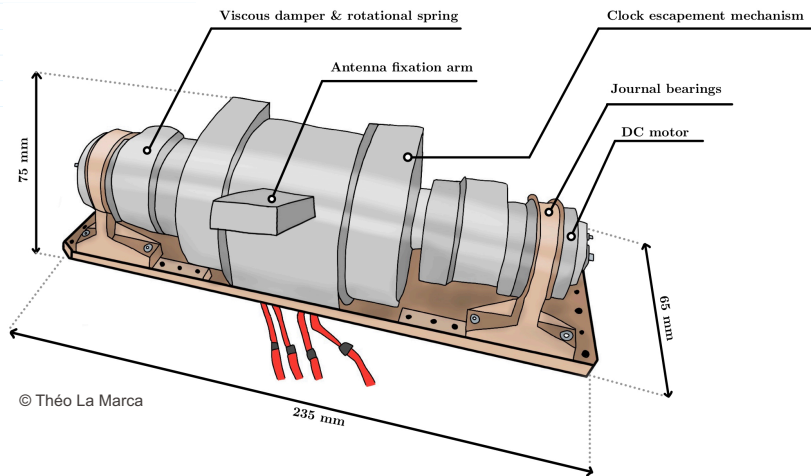
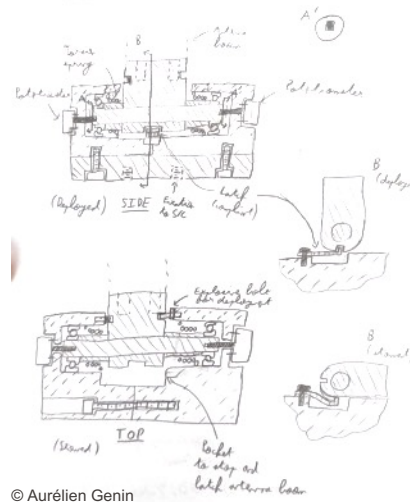


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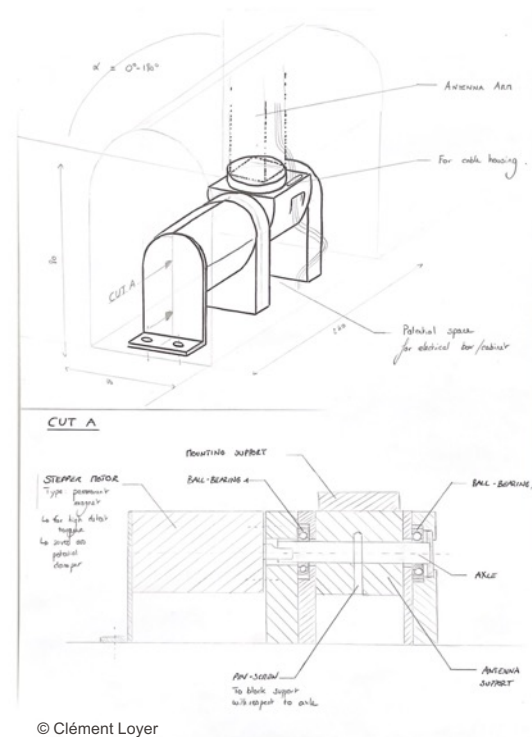
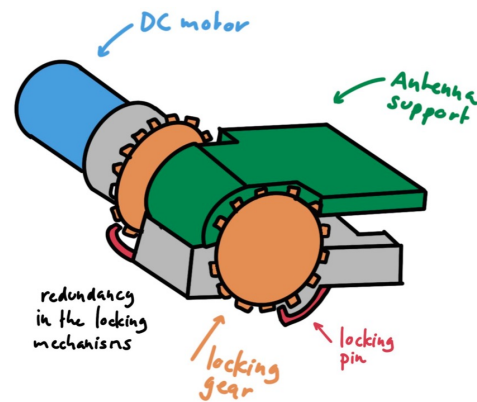
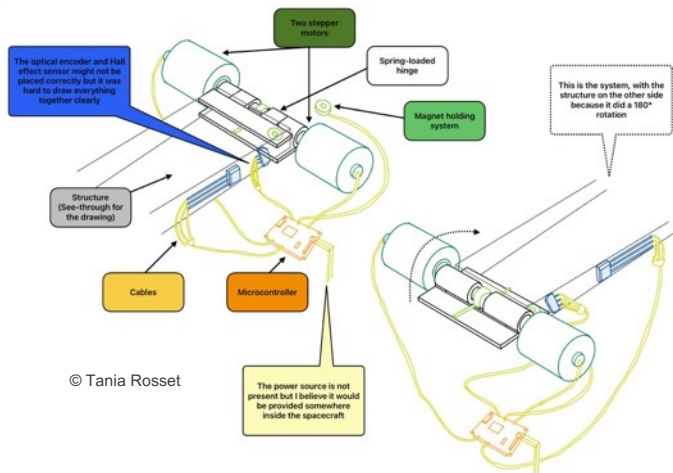
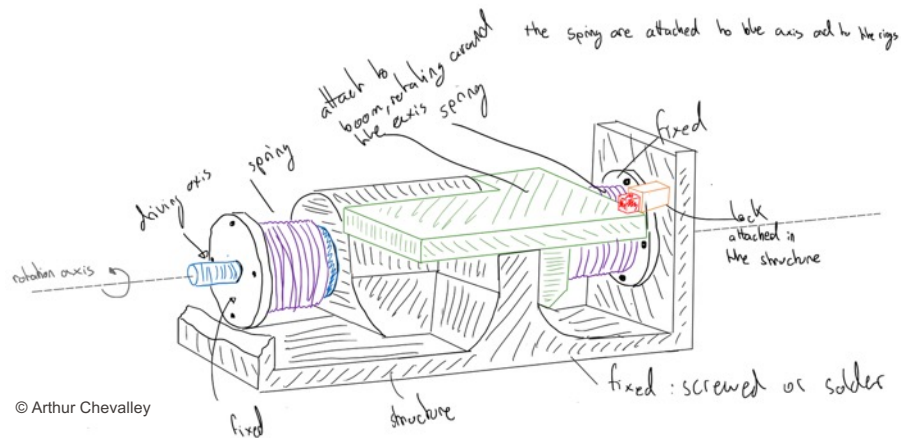




- 1 main shaft - hollow on left for cable passage (& weight)
- 2 electric motor
- 3 planetary reduction gears  $\rightarrow$  reduction chosen to be self-locking
- 4 electromagnetic clutch  $\rightarrow$  default "engaged"
- 5 16-bit encoder
- 6 oblique ball bearings
- 7 spring to preload mechanism
- 8 accordion connector - allow for thermal expansion/contractions



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# Mini Project Summary

- Identify the critical and key requirements
- List the functions
- Map the functions to forms (architecture)
  - List of potential components, incl. specifications
- Make preliminary analysis
  - What are the required performances?
  - Pre-sizing of the components
- Selection of concept and components
  - E.g. Pugh matrix, ...
- Sketch your concept(s)
- Make some preliminary budgets (size, mass, power, ...)
- Convince customer, management, ... that you will be able to achieve the requirements, within time and budget

- Theme 7 – Reliability
- Fill the exam schedule on MOODLE (Exams June 27 & 30, July 1)  
<https://moodle.epfl.ch/mod/scheduler/view.php?id=1206907>  
**To be filled until June 6<sup>th</sup>, 17:00.**